

Lecture 16 – ME6402, Spring 2025

Feedback Linearization

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Goals of Lecture 16

- ▶ Relative degree
- ▶ Input-output linearization
- ▶ Zero dynamics

Additional Reading

- ▶ Khalil Chapter 13
- ▶ Sastry Chapter 9

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Relative Degree

Today: Relative degree, input-output linearization, zero dynamics

Consider the single-input single-output (SISO) nonlinear system:

$$\begin{aligned}\dot{x} &= f(x) + g(x)u \\ y &= h(x).\end{aligned}\tag{1}$$

Relative degree (informal definition): Number of times we need to take the time derivative of the output to see the input:

$$\begin{aligned}\dot{y} &= \underbrace{\frac{\partial h}{\partial x} f(x)} + \underbrace{\frac{\partial h}{\partial x} g(x)} u \\ &=: L_f h(x) \quad =: L_g h(x)\end{aligned}$$

- ▶ $L_f h$ is called the *Lie derivative* of h along the vector field f

Relative Degree (cont.)

If $L_g h(x) \neq 0$ in an open set containing the equilibrium, then the relative degree is equal to 1. If $L_g h(x) \equiv 0$, continue taking derivatives:

$$\begin{aligned}\ddot{y} &= \underbrace{L_f L_f h(x)} + L_g L_f h(x)u. \\ &=: L_f^2 h(x)\end{aligned}$$

If $L_g L_f h(x) \neq 0$, then relative degree is 2. If $L_g L_f h(x) \equiv 0$, continue.

Relative Degree (cont.)

Definition: The system (2) has *relative degree* r if, in a neighbourhood of the equilibrium,

$$L_g L_f^{i-1} h(x) = 0 \quad i = 1, 2, \dots, r-1$$

$$L_g L_f^{r-1} h(x) \neq 0.$$

$$\begin{aligned} \dot{x} &= f(x) + g(x)u \\ y &= h(x). \end{aligned} \quad (2)$$

Example 1

The system

$$\dot{x}_1 = x_2$$

$$\dot{x}_2 = -x_1^3 + u$$

$$y = x_1$$

has relative degree

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The system

$$\dot{x}_1 = x_2$$

$$\dot{x}_2 = -x_1^3 + u$$

$$y = x_1$$

has relative degree = 2.

Example 2

SISO linear system:

$$\dot{x} = Ax + Bu \quad y = Cx$$

$$L_g h(x) = CB, \quad L_g L_f h(x) = CAB, \quad \dots, \quad L_g L_f^{r-1} h(x) = CA^{r-1}B.$$

- ▶ $CB \neq 0 \Rightarrow$ relative degree = 1
- ▶ $CB = 0, \quad CAB \neq 0 \Rightarrow$ relative degree = 2
- ▶ $CB = \dots = CA^{r-2}B = 0, \quad CA^{r-1}B \neq 0 \Rightarrow$ relative degree = r

The parameters $CA^{i-1}B$ $i = 1, 2, 3, \dots$ are called *Markov parameters* and are invariant under similarity transformations.

Example 3

$$\dot{x}_1 = x_2 + x_3^3 \quad y = x_1$$

$$\dot{x}_2 = x_3$$

$$\dot{x}_3 = u$$

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$$\dot{x}_1 = x_2 + x_3^3 \quad y = x_1$$

$$\dot{x}_2 = x_3 \quad \dot{y} = \dot{x}_1 = x_2 + x_3^3$$

$$\dot{x}_3 = u \quad \ddot{y} = \dot{x}_2 + 3x_3^2\dot{x}_3 = x_3 + 3x_3^2u$$

$L_g L_f h(x) = 3x_3^2 = 0$ when $x_3 = 0$, and $\neq 0$ elsewhere. Thus, this system does not have a well-defined relative degree around $x = 0$.

Input-Output Linearization

If a system has a well-defined relative degree then it is input-output linearizable:

$$y^{(r)} = L_f^r h(x) + \underbrace{L_g L_f^{r-1} h(x)}_{\neq 0} u$$

Apply preliminary feedback:

$$u = \frac{1}{L_g L_f^{r-1} h(x)} \left(-L_f^r h(x) + v \right) \quad (3)$$

where v is a new input to be designed.

Input-Output Linearization (cont.)

Then, $y^{(r)} = v$ is a linear system in the form of an integrator chain:

$$\dot{\zeta}_1 = \zeta_2$$

$$\dot{\zeta}_2 = \zeta_3$$

$$\vdots$$

$$\dot{\zeta}_r = v$$

where $\zeta_1 =: y = h(x)$, $\zeta_2 =: \dot{y} = L_f h(x)$, \dots , $\zeta_r =: y^{(r-1)} = L_f^{r-1} h(x)$.



$$y^{(r)} = L_f^r h(x) + \underbrace{L_g L_f^{r-1} h(x)}_{\neq 0} u$$



$$u = \frac{1}{L_g L_f^{r-1} h(x)} \cdot \left(-L_f^r h(x) + v \right)$$

Input-Output Linearization (cont.)

To ensure $y(t) \rightarrow 0$ as $t \rightarrow \infty$, apply the feedback:

$$\begin{aligned} v &= -k_1 \zeta_1 - k_2 \zeta_2 - \dots - k_r \zeta_r \\ &= -k_1 h(x) - k_2 L_f h(x) - \dots - k_r L_f^{r-1} h(x) \end{aligned} \quad (4)$$

where k_1, \dots, k_r are such that $s^r + k_r s^{r-1} + \dots + k_2 s + k_1$ has all roots in the open left half-plane.



$$\dot{\zeta}_1 = \zeta_2$$

$$\dot{\zeta}_2 = \zeta_3$$

$$\vdots$$

$$\dot{\zeta}_r = v$$

Zero Dynamics

Does the controller (5)-(6) achieve asymptotic stability of $x = 0$?

Not necessarily! It renders the $(n - r)$ -dimensional manifold:

$$h(x) = L_f h(x) = \dots = L_f^{r-1} h(x) = 0$$

invariant and attractive.

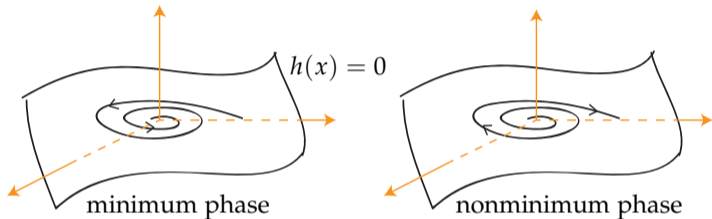
- ▶ The dynamics restricted to this manifold are called zero dynamics and determine whether or not $x = 0$ is stable.
- ▶ If the origin of the zero dynamics is asymptotically stable, the system is called minimum phase. If unstable, it is called nonminimum phase.

$$u = \frac{1}{L_g L_f^{r-1} h(x)} \left(-L_f^r h(x) + v \right) \quad (5)$$

$$\begin{aligned} v &= -k_1 \zeta_1 - k_2 \zeta_2 - \dots - k_r \zeta_r \\ &= -k_1 h(x) - k_2 L_f h(x) - \\ &\quad \dots - k_r L_f^{r-1} h(x) \end{aligned} \quad (6)$$

Zero Dynamics (cont.)

Example: $n = 3$, $r = 1$



Finding the Zero Dynamics

Set $y = \dot{y} = \dots = y^{(r-1)} = 0$ and substitute (5) with $v = 0$, that is:

$$u^* = \frac{-L_f^r h(x)}{L_g L_f^{r-1} h(x)}.$$

The remaining dynamical equations describe the zero dynamics.

Finding the Zero Dynamics: Example

Example:

$$\begin{aligned}\dot{x}_1 &= x_2 \\ \dot{x}_2 &= \alpha x_3 + u \\ \dot{x}_3 &= \beta x_3 - u \\ y &= x_1\end{aligned}\tag{7}$$

This system has relative degree 2. With $x_1 = x_2 = 0$ and $u^* = -\alpha x_3$, the remaining dynamical equation is

$$\dot{x}_3 = (\alpha + \beta)x_3.$$

Thus this system is minimum phase if $\alpha + \beta < 0$.

Zero Dynamics of a Linear System

For a linear SISO system, *relative degree* is the difference between the degrees of the denominator and the numerator of the transfer function, and *zeros* are the roots of the numerator. The definitions of relative degree and zero dynamics above generalize these concepts to nonlinear systems.

As an example, the transfer function for (8) is

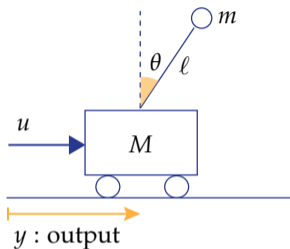
$$\frac{s - (\alpha + \beta)}{s^2(s - \beta)},$$

which has relative degree two and a zero at $s = \alpha + \beta$ as expected.

$$\begin{aligned}\dot{x}_1 &= x_2 \\ \dot{x}_2 &= \alpha x_3 + u \\ \dot{x}_3 &= \beta x_3 - u \\ y &= x_1\end{aligned}\tag{8}$$

Example

Example: Cart/Pole



$$\ddot{y} = \frac{1}{\frac{M}{m} + \sin^2 \theta} \left(\frac{u}{m} + \dot{\theta}^2 \ell \sin \theta - g \sin \theta \cos \theta \right)$$

$$\ddot{\theta} = \frac{1}{\ell \left(\frac{M}{m} + \sin^2 \theta \right)} \left(-\frac{u}{m} \cos \theta - \dot{\theta}^2 \ell \cos \theta \sin \theta + \frac{M+m}{m} g \sin \theta \right)$$

Relative degree = 2.

Example (cont.)

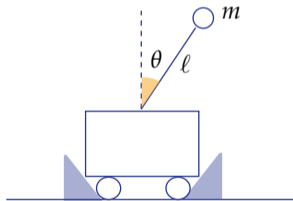
To find the zero dynamics, substitute $y = \dot{y} = 0$, and

$$u^* = -m(\dot{\theta}^2 \ell \sin \theta - g \sin \theta \cos \theta)$$

in the $\ddot{\theta}$ equation:

$$\ddot{\theta} = \frac{g}{\ell} \sin \theta.$$

Same as the dynamics of the pole when the cart is held still:



Nonminimum phase because $\theta = 0$ is unstable for the zero dynamics.

$$\ddot{y} = \frac{1}{\frac{M}{m} + \sin^2 \theta} \cdot \left(\frac{u}{m} + \dot{\theta}^2 \ell \sin \theta - g \sin \theta \cos \theta \right)$$
$$\ddot{\theta} = \frac{1}{\ell \left(\frac{M}{m} + \sin^2 \theta \right)} \cdot \left(-\frac{u}{m} \cos \theta - \dot{\theta}^2 \ell \cos \theta \sin \theta + \frac{M+m}{m} g \sin \theta \right)$$