

# ME 6402 – Lecture 16

## FEEDBACK LINEARIZATION 1

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Overview:

- Relative Degree
- Input-Output Linearization
- Zero Dynamics

Additional Reading

- Khalil 13.2

### Relative Degree

Consider the single-input single-output (SISO) nonlinear system:

$$\begin{aligned}\dot{x} &= f(x) + g(x)u \\ y &= h(x).\end{aligned}\tag{1}$$

with  $y \in \mathbb{R}$  and  $u \in \mathbb{R}$ .

Relative degree (informal definition): Number of times we need to take the time derivative of the output to see the input:

$L_f h$  is called the *Lie derivative* of  $h$  along the vector field  $f$

$$\begin{aligned}\dot{y} &= \frac{dh(x)}{dt} \\ &= \frac{\partial h(x)}{\partial x} \frac{\partial x}{\partial t} \\ &= \underbrace{\frac{\partial h}{\partial x} f(x)}_{=: L_f h(x)} + \underbrace{\frac{\partial h}{\partial x} g(x)}_{=: L_g h(x)} u \\ &=: L_f h(x) + L_g h(x) u\end{aligned}$$

If  $L_g h(x) \neq 0$  in an open set containing the equilibrium, then the relative degree is equal to 1. If  $L_g h(x) \equiv 0$ , continue taking derivatives:

$$\begin{aligned}\ddot{y} &= \underbrace{L_f L_f h(x)}_{=: L_f^2 h(x)} + L_g L_f h(x) u.\end{aligned}$$

If  $L_g L_f h(x) \neq 0$ , then relative degree is 2. If  $L_g L_f h(x) \equiv 0$ , continue.

**Definition: Relative Degree.** The system (1) has *relative degree*  $r$  if, in a neighbourhood of the equilibrium,

$$\begin{aligned}L_g L_f^{i-1} h(x) &= 0 \quad i = 1, 2, \dots, r-1 \\ L_g L_f^{r-1} h(x) &\neq 0.\end{aligned}\tag{2}$$

Examples:

$$\begin{aligned}
 1. \quad \dot{x}_1 &= x_2 \\
 \dot{x}_2 &= -x_1^3 + u \\
 y &= x_1
 \end{aligned} \tag{3}$$

has relative degree = 2 as shown below:

$$\begin{aligned}
 y &= x_1 \\
 \dot{y} &= \dot{x}_1 = x_2 \\
 \ddot{y} &= \dot{x}_2 = -x_1^3 + u
 \end{aligned}$$

2. SISO linear system:

$$\dot{x} = Ax + Bu \quad y = Cx$$

$$L_g h(x) = CB, \quad L_g L_f h(x) = CAB, \quad \dots, \quad L_g L_f^{r-1} h(x) = CA^{r-1} B.$$

$$CB \neq 0 \Rightarrow \text{relative degree} = 1$$

$$CB = 0, \quad CAB \neq 0 \Rightarrow \text{relative degree} = 2$$

$$CB = \dots = CA^{r-2} B = 0, \quad CA^{r-1} B \neq 0 \Rightarrow \text{relative degree} = r$$

The parameters  $CA^{i-1} B$   $i = 1, 2, 3, \dots$  are called *Markov parameters* and are invariant under similarity transformations.

$$\begin{aligned}
 3. \quad \dot{x}_1 &= x_2 + x_3^3 & y &= x_1 \\
 \dot{x}_2 &= x_3 & \dot{y} &= \dot{x}_1 = x_2 + x_3^3 \\
 \dot{x}_3 &= u & \ddot{y} &= \dot{x}_2 + 3x_3^2 \dot{x}_3 = x_3 + 3x_3^2 u
 \end{aligned}$$

$L_g L_f h(x) = 3x_3^2 = 0$  when  $x_3 = 0$ , and  $\neq 0$  elsewhere. Thus, this system does not have a well-defined relative degree around  $x = 0$ .

*Input-Output Linearization*

If a system has a well-defined relative degree then it is input-output linearizable:

$$y^{(r)} = L_f^r h(x) + \underbrace{L_g L_f^{r-1} h(x)}_{\neq 0} u$$

Apply preliminary feedback:

$$u = \frac{1}{L_g L_f^{r-1} h(x)} \left( -L_f^r h(x) + v \right) \tag{4}$$

where  $v$  is a new input to be designed. Then,  $y^{(r)} = v$  is a linear system in the form of an integrator chain:

$$\begin{aligned}\dot{\zeta}_1 &= \zeta_2 \\ \dot{\zeta}_2 &= \zeta_3 \\ &\vdots \\ \dot{\zeta}_r &= v\end{aligned}$$

where  $\zeta_1 =: y = h(x)$ ,  $\zeta_2 =: \dot{y} = L_f h(x)$ ,  $\dots$ ,  $\zeta_r =: y^{(r-1)} = L_f^{r-1} h(x)$ . To ensure  $y(t) \rightarrow 0$  as  $t \rightarrow \infty$ , apply the feedback:

$$\begin{aligned}v &= -k_1 \zeta_1 - k_2 \zeta_2 - \dots - k_r \zeta_r \\ &= -k_1 h(x) - k_2 L_f h(x) - \dots - k_r L_f^{r-1} h(x)\end{aligned}\quad (5)$$

where  $k_1, \dots, k_r$  are such that  $s^r + k_r s^{r-1} + \dots + k_2 s + k_1$  has all roots in the open left half-plane.

Does the controller (4)-(5) achieve asymptotic stability of  $x = 0$ ?

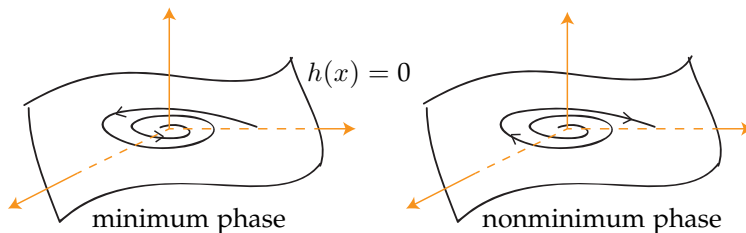
Not necessarily! It renders the  $(n - r)$ -dimensional manifold:

$$h(x) = L_f h(x) = \dots = L_f^{r-1} h(x) = 0$$

invariant and attractive. The dynamics restricted to this manifold are called zero dynamics and determine whether or not  $x = 0$  is stable.

If the origin of the zero dynamics is asymptotically stable, the system is called minimum phase. If unstable, it is called nonminimum phase.

Example:  $n = 3$ ,  $r = 1$



### Finding the Zero Dynamics

Set  $y = \dot{y} = \dots = y^{(r-1)} = 0$  and substitute (4) with  $v = 0$ , that is:

$$u^* = \frac{-L_f^r h(x)}{L_g L_f^{r-1} h(x)}.$$

The remaining dynamical equations describe the zero dynamics.

Example:

$$\begin{aligned}\dot{x}_1 &= x_2 \\ \dot{x}_2 &= \alpha x_3 + u \\ \dot{x}_3 &= \beta x_3 - u \\ y &= x_1\end{aligned}\quad (6)$$

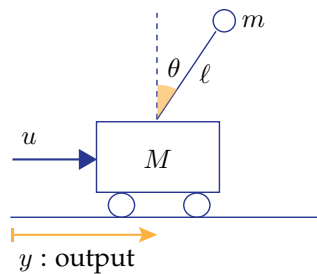
This system has relative degree 2. With  $x_1 = x_2 = 0$  and  $u^* = -\alpha x_3$ , the remaining dynamical equation is

$$\dot{x}_3 = (\alpha + \beta)x_3.$$

Thus this system is minimum phase if  $\alpha + \beta < 0$ .

Example: Cart/Pole <sup>1</sup>

<sup>1</sup> Simulation code for this example is available [online](#)



$$\begin{aligned}\ddot{y} &= \frac{1}{\frac{M}{m} + \sin^2 \theta} \left( \frac{u}{m} + \dot{\theta}^2 \ell \sin \theta - g \sin \theta \cos \theta \right) \\ \ddot{\theta} &= \frac{1}{\ell \left( \frac{M}{m} + \sin^2 \theta \right)} \left( -\frac{u}{m} \cos \theta - \dot{\theta}^2 \ell \cos \theta \sin \theta + \frac{M+m}{m} g \sin \theta \right)\end{aligned}\quad (7)$$

Relative degree = 2.

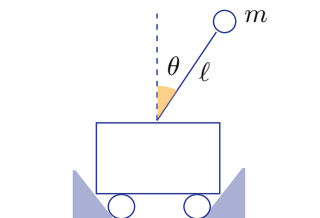
To find the zero dynamics, substitute  $y = \dot{y} = 0$ , and

$$u^* = -m(\dot{\theta}^2 \ell \sin \theta - g \sin \theta \cos \theta)$$

in the  $\ddot{\theta}$  equation:

$$\ddot{\theta} = \frac{g}{\ell} \sin \theta.$$

Same as the dynamics of the pole when the cart is held still:



Nonminimum phase because  $\theta = 0$  is unstable for the zero dynamics.

*Appendix: Derivation of Cart-Pole Equations of Motion*

We will define our state to be  $x = [q, \dot{q}]$  with  $q = [p, \theta]$  for  $p$  being the horizontal position of the cart and  $\theta$  being the angle of the pole with respect to the vertical. The equations of motion are derived using the Euler-Lagrange equation.

First, we can compute the kinetic energy of our system in terms of the cart's energy and the pendulum's energy:

$$\begin{aligned}
 K_{\text{cart}} &= \frac{1}{2}M\dot{p}^2 \\
 K_{\text{pend}} &= \frac{1}{2}m(\dot{x}_p^2 + \dot{y}_p^2) \\
 &= \frac{1}{2}m((\dot{p} + \ell \cos(\theta)\dot{\theta})^2 + (\ell \sin(\theta)\dot{\theta})^2) \\
 &= \frac{1}{2}m(\dot{p}^2 + 2\ell \cos(\theta)\dot{p}\dot{\theta} + \ell^2 \cos^2(\theta)\dot{\theta}^2 + \ell^2 \sin^2(\theta)\dot{\theta}^2) \\
 &= \frac{1}{2}m(\dot{p}^2 + 2\ell \cos(\theta)\dot{p}\dot{\theta} + \ell^2\dot{\theta}^2) \\
 K_{\text{total}} &= K_{\text{cart}} + K_{\text{pend}} \\
 &= \frac{1}{2}(M + m)\dot{p}^2 + m\ell \cos(\theta)\dot{p}\dot{\theta} + \frac{1}{2}m\ell^2\dot{\theta}^2
 \end{aligned}$$

The simplification in  $K_{\text{pend}}$  comes from  $\cos^2 + \sin^2 = 1$

Next, we can compute the potential energy of the two subsystems as:

$$\begin{aligned}
 P_{\text{cart}} &= 0 \\
 P_{\text{pend}} &= -mg\ell \cos(\theta) \\
 P_{\text{total}} &= P_{\text{cart}} + P_{\text{pend}} = -mg\ell \cos(\theta)
 \end{aligned}$$

Plugging these into our Euler-Lagrange Equation with  $L = K - P$  we

get:

$$\frac{d}{dt} \left( \left[ \begin{array}{c} \frac{\partial(\frac{1}{2}(M+m)\dot{p}^2 + m\ell \cos(\theta)\dot{p}\dot{\theta} + \frac{1}{2}m\ell^2\dot{\theta}^2 + mgl \cos(\theta))}{\partial \dot{p}} \\ \frac{\partial(\frac{1}{2}(M+m)\dot{p}^2 + m\ell \cos(\theta)\dot{p}\dot{\theta} + \frac{1}{2}m\ell^2\dot{\theta}^2 + mgl \cos(\theta))}{\partial \dot{\theta}} \end{array} \right] \right) - \left[ \begin{array}{c} \frac{\partial(\frac{1}{2}(M+m)\dot{p}^2 + m\ell \cos(\theta)\dot{p}\dot{\theta} + \frac{1}{2}m\ell^2\dot{\theta}^2 + mgl \cos(\theta))}{\partial p} \\ \frac{\partial(\frac{1}{2}(M+m)\dot{p}^2 + m\ell \cos(\theta)\dot{p}\dot{\theta} + \frac{1}{2}m\ell^2\dot{\theta}^2 + mgl \cos(\theta))}{\partial \theta} \end{array} \right] = \begin{bmatrix} u \\ 0 \end{bmatrix}$$

$$\frac{d}{dt} \left( \left[ \begin{array}{c} (M+m)\dot{p} + m\ell \cos(\theta)\dot{\theta} \\ m\ell \cos(\theta)\dot{p} + m\ell^2\dot{\theta} \end{array} \right] \right) - \left[ \begin{array}{c} 0 \\ -m\ell \sin(\theta)\dot{p}\dot{\theta} - mgl \sin(\theta) \end{array} \right] = \begin{bmatrix} u \\ 0 \end{bmatrix}$$

$$\left[ \begin{array}{c} (M+m)\ddot{p} - m\ell \sin(\theta)\dot{\theta}^2 + m\ell \cos(\theta)\ddot{\theta} \\ -m\ell \sin(\theta)\dot{\theta}\dot{p} + m\ell \cos(\theta)\ddot{p} + m\ell^2\ddot{\theta} \end{array} \right] - \left[ \begin{array}{c} 0 \\ -m\ell \sin(\theta)\dot{p}\dot{\theta} - mgl \sin(\theta) \end{array} \right] = \begin{bmatrix} u \\ 0 \end{bmatrix}$$

$$\left[ \begin{array}{c} (M+m)\ddot{p} - m\ell \sin(\theta)\dot{\theta}^2 + m\ell \cos(\theta)\ddot{\theta} \\ m\ell \cos(\theta)\ddot{p} + m\ell^2\ddot{\theta} + mgl \sin(\theta) \end{array} \right] = \begin{bmatrix} u \\ 0 \end{bmatrix}$$

This can be rearranged to separate  $\ddot{p}$  and  $\ddot{\theta}$ :

$$\underbrace{\begin{bmatrix} M+m & m\ell \cos(\theta) \\ m\ell \cos(\theta) & m\ell^2 \end{bmatrix}}_M \underbrace{\begin{bmatrix} \ddot{p} \\ \ddot{\theta} \end{bmatrix}} + \underbrace{\begin{bmatrix} -m\ell \sin(\theta)\dot{\theta}^2 \\ mgl \sin(\theta) \end{bmatrix}}_H = \underbrace{\begin{bmatrix} 1 \\ 0 \end{bmatrix}}_B u$$

Note that this actually follows our standard robotic equations of motion  $(M(q)\ddot{q} + H(q, \dot{q}) = Bu)$  so we can follow the standard procedure:

$$\ddot{q} = M^{-1}(-H(q, \dot{q}) + Bu)$$

Plugging all of this into a symbolic solver, we arrive at our previous equations of motion:

$$\ddot{p} = \frac{1}{\frac{M}{m} + \sin^2 \theta} \left( \frac{u}{m} + \dot{\theta}^2 \ell \sin \theta - g \sin \theta \cos \theta \right)$$

$$\ddot{\theta} = \frac{1}{\ell \left( \frac{M}{m} + \sin^2 \theta \right)} \left( -\frac{u}{m} \cos \theta - \dot{\theta}^2 \ell \cos \theta \sin \theta + \frac{M+m}{m} g \sin \theta \right)$$